

# Some considerations on DNS and LES in complex geometries

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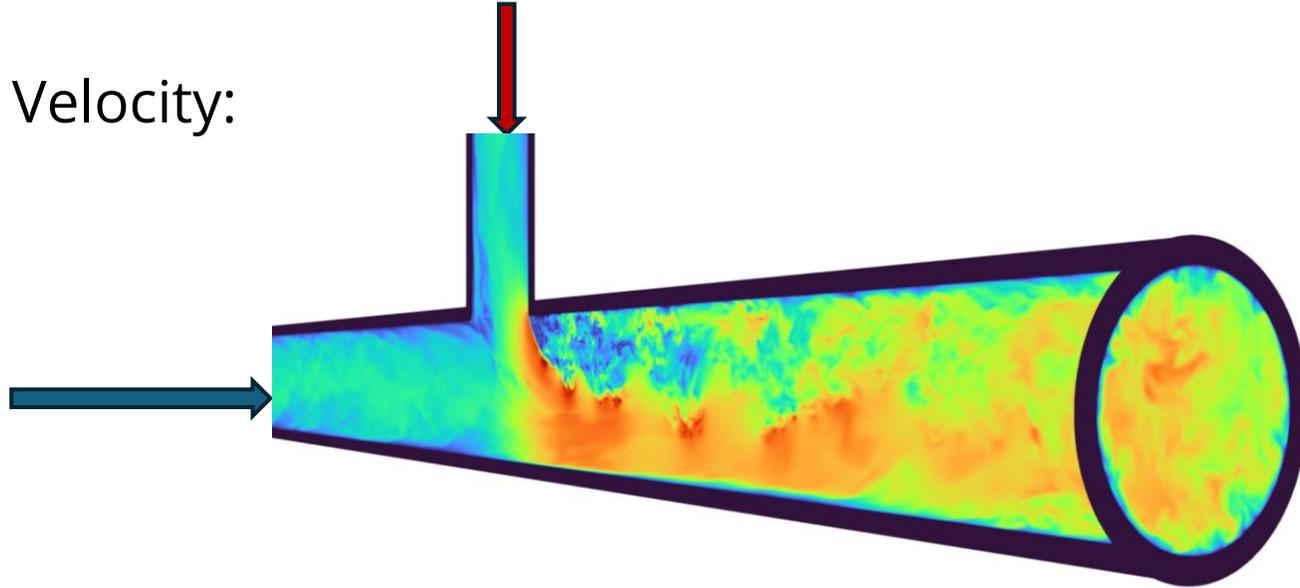
# Contents

## High-fidelity (HiFi) simulation in complex geometries

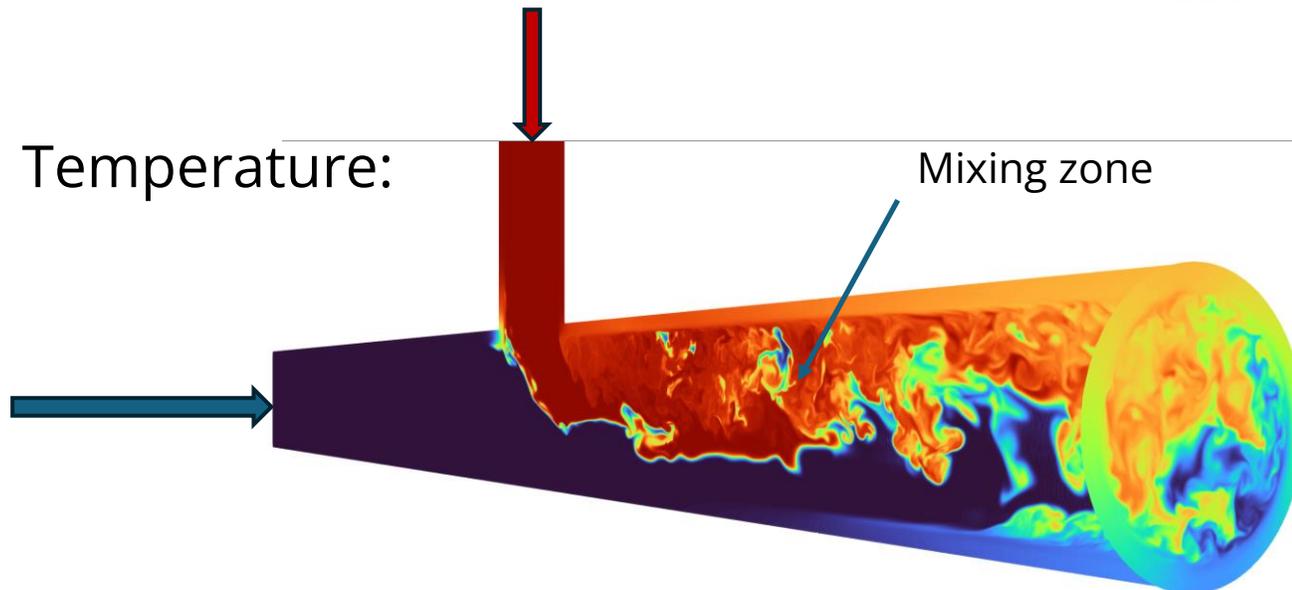
- Single-phase flow HiFi simulations
  - DNS
  - LES
- Two-phase flow HiFi simulations
  - Ambition
  - Numerical challenges
  - Modelling of sub-grid physics
- Concluding remarks

# HiFi single-phase simulations - DNS

- Velocity:



- Temperature:

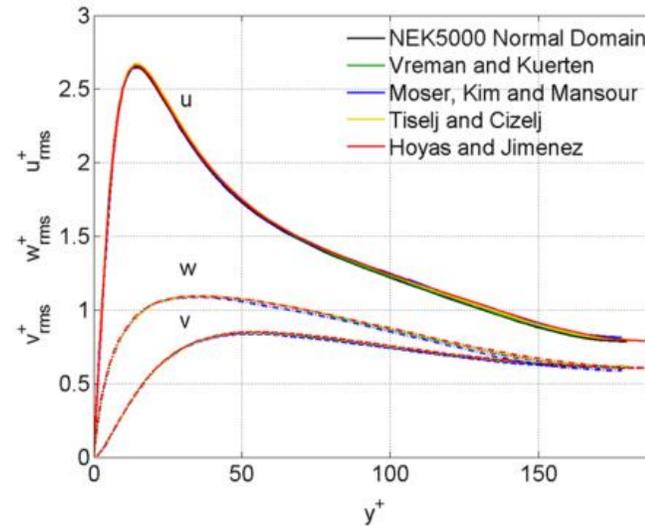
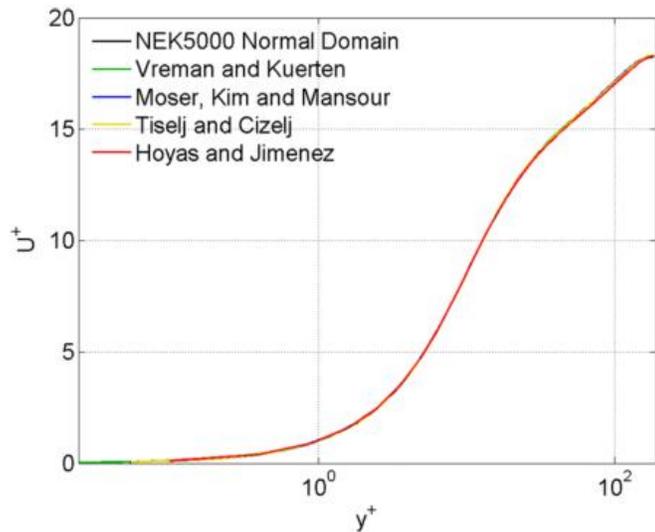


*A DNS example:*

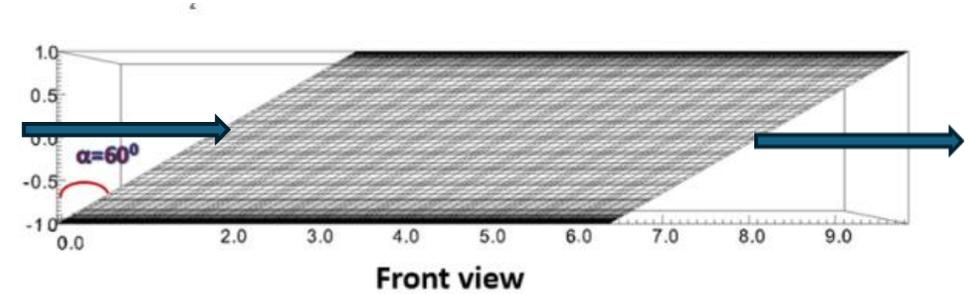
- Thermal-fatigue issue
- NEK-RS spectral element method
  - Lagrange interpolants on a Gauss-Lobatto-Legendre points distribution
- Purpose: knowledge and reference data for turbulence model development & validation

# HiFi single-phase simulations - DNS

- NEK-5000 / NEK-RS spectral element method
  - High-order accuracy in complex geometries<sup>1</sup>
    - DNS of turbulent channel flow



Mean and RMS velocities



<sup>1</sup>Shams and Komen *FTAC*, 2018

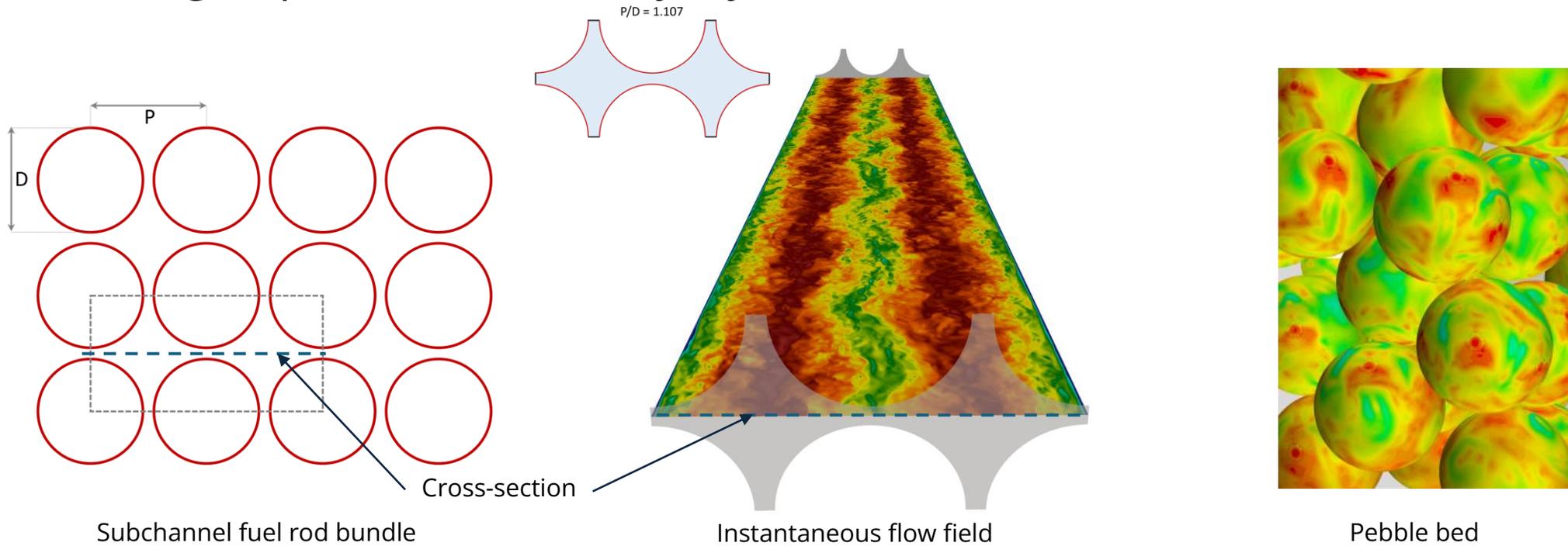
# HiFi single-phase simulations - DNS

- NEK-RS / NEK5000 spectral element method
  - Accurate, yes! Efficient?
  - Costs vs accuracy assessments:
    - <sup>1</sup>Kooij et al, *Computers & Fluids*, 2018
      - Rayleigh–Bénard (RB) convection in cubic and cylindrical containers
    - <sup>2</sup>Capuano et al, *Eur. J. Mech. B Fluids*, 2023
      - Turbulent flow over a sphere
  - Conclusion: NEK5000 less efficient<sup>1</sup> than or similarly efficient<sup>2</sup> as Cartesian / cylindrical grid finite difference / volume solvers using FFTs.

# HiFi single-phase simulations - DNS

- NEK-RS / NEK5000 spectral element method
  - Efficiency: depends on Re number and geometry<sup>2</sup>
    - IBM needs refinement in multiple coordinate directions for clustering grid points in boundary layer → Cost increases fast with Re number

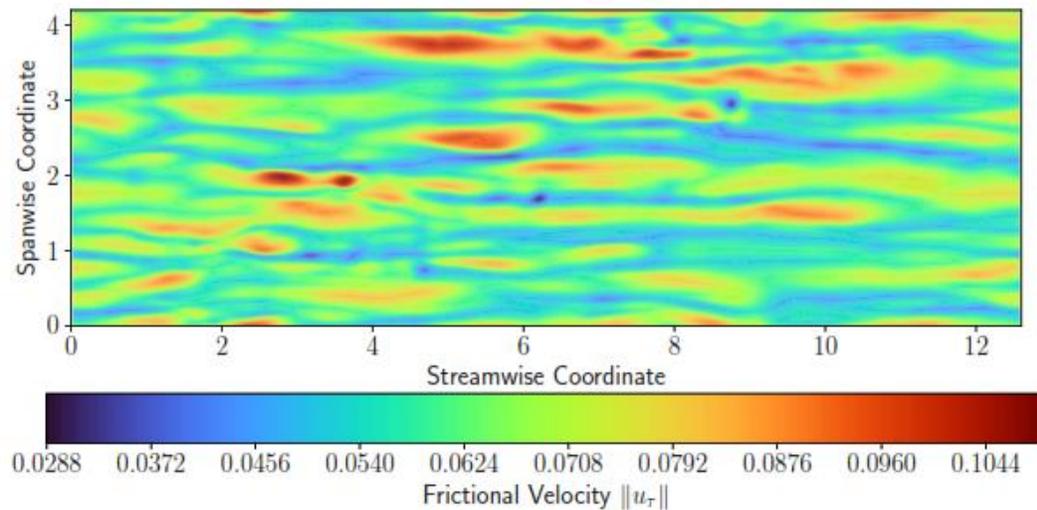
<sup>2</sup>Capuano et al, *Eur. J. Mech. B Fluids*, 2023



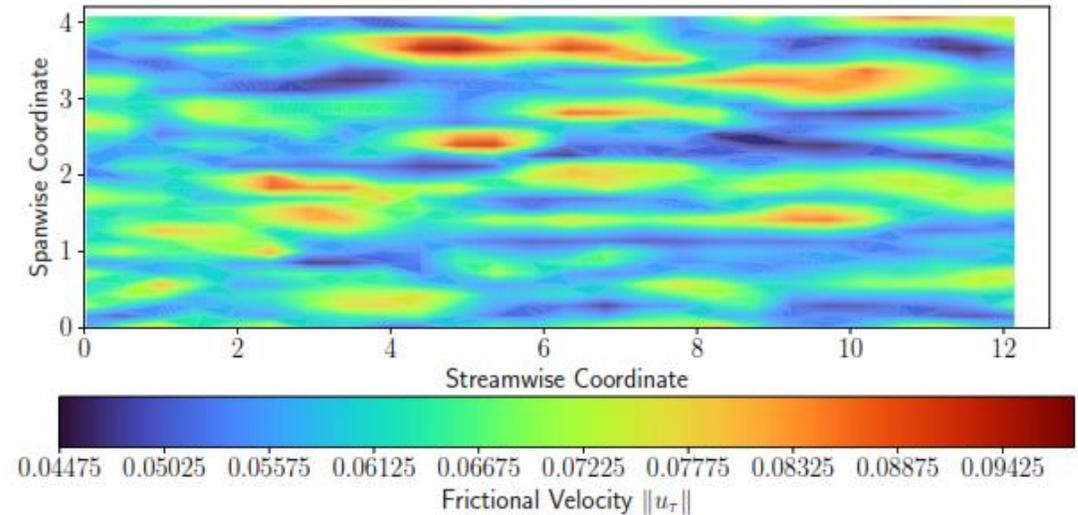
- Comparison higher-order moments

# HiFi single-phase simulations - LES

- LES: effect of implicit top-hat filter
  - DNS turbulent channel flow data  $\rightarrow$  top-hat filtering to match UDNS grid resolution, with the same wall normal resolution in DNS and UDNS resolution\*



Friction velocity, DNS resolution



Filtered friction velocity, UDNS resolution

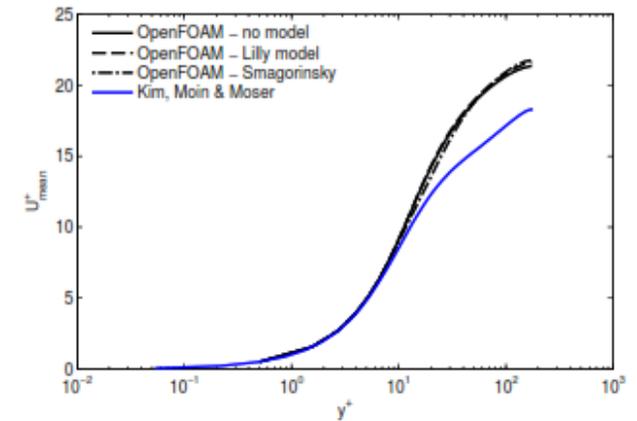
Top-hat filter  $\rightarrow$  projection flow topology at lower Reynolds

\* Mathur and Sajuhdeen

# HiFi single-phase simulations - LES

- LES: effect of implicit top-hat filter
  - DNS turbulent channel flow data → top-hat filtering to match UDNS grid resolution, with the same wall normal resolution in DNS and UDNS resolution

Case	$Re_\tau$ scaling				$Re_{bulk}$ scaling			
	$u_\tau$	$U_{bulk}$	$Re_\tau$	$Re_{bulk}$	$u_\tau$	$U_{bulk}$	$Re_\tau$	$Re_{bulk}$
Vreman	1.000	15.70	180.0	5650	1.000	15.70	180.0	5650
OpenFOAM 30-Grid	1.007	15.68	181.3	5643	0.998	15.66	179.6	5638
OpenFOAM 60-Grid	1.007	17.24	181.3	6205	0.924	15.66	166.3	5637
OpenFOAM 90-Grid	1.000	18.68	179.9	6723	0.839	15.66	150.9	5637



(a) 90-Grid - mean velocity

Frictional and bulk Reynolds number for Vreman DNS data and 3 OpenFOAM UDNS simulations\*

- Top-hat filtering → projection at lower Reynolds flow topology
- LES SGS viscosity (algebraic models) → additional projection
- Issue: engineers are not interested in a filtered reality

\*Komen, Tue PhD thesis, 2025

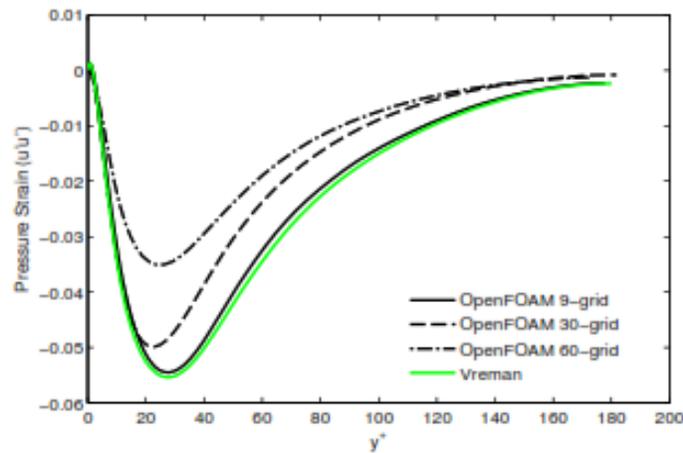
# HiFi single-phase simulations - LES

- LES: issue with algebraic models for wall resolved LES

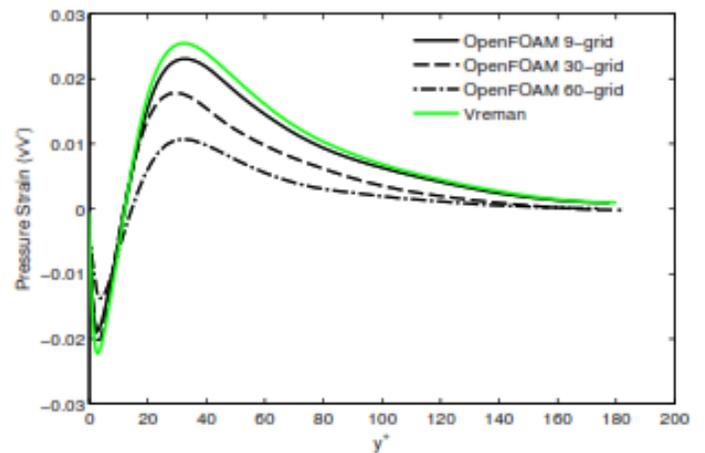
- Case: DNS and UDNS of turbulent channel flow<sup>1</sup>

<sup>1</sup>Komen et al, *JCP*, 2017

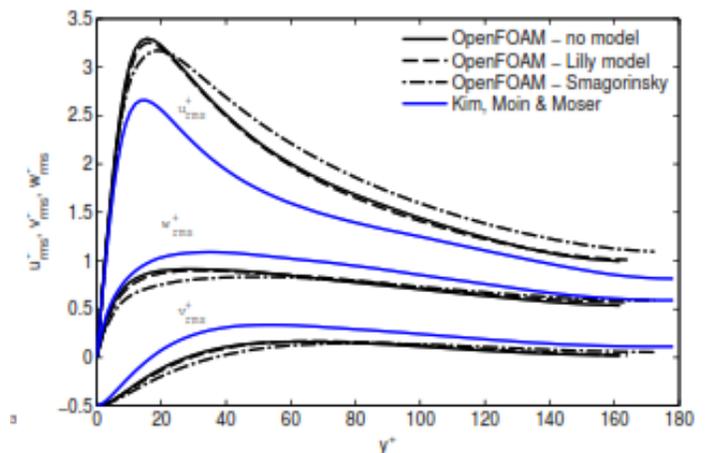
- Systematic coarsening of the UDNS grid resolution gives:
  - progressive underprediction of pressure strain rate terms →
  - progressive underprediction of redistribution of kinetic energy →
  - progressive overprediction of  $u'u'$ , and underpredictions of  $v'v'$  and  $w'w'$



(b) Pressure strain rate of  $\overline{u'v'}$



(d) Pressure strain rate of  $\overline{v'v'}$



(d) 60-Grid - RMS velocities

# High-Fidelity single-phase simulations

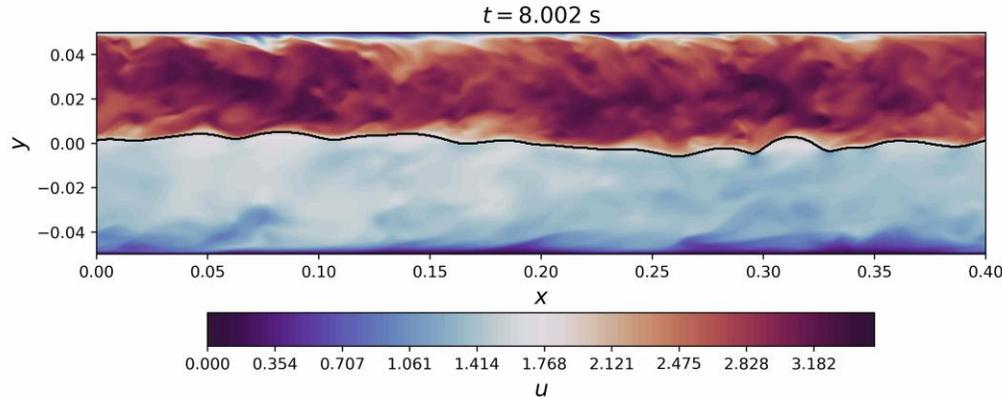
- LES: issue with algebraic models for wall resolved LES
  - Case: DNS and UDNS of turbulent channel flow<sup>1</sup>
    - Issue: algebraic LES models cannot compensate for this deficiency in redistribution of turbulent kinetic energy!
    - Instead, they need grid resolutions approaching DNS grid resolutions
      - for which LES model contribution becomes negligible
    - Isotropy assumption of the SGS stresses does not hold in wall resolved simulations → (probably) anisotropic models needed
    - How to predict the correct friction velocity?

<sup>1</sup>Komen et al, *JCP*, 2017

# HiFi two-phase simulations - DNS

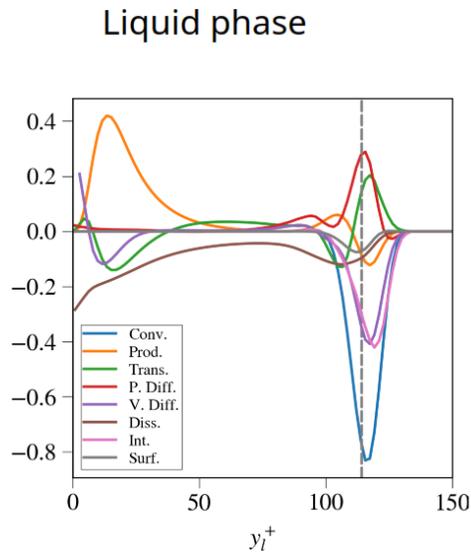
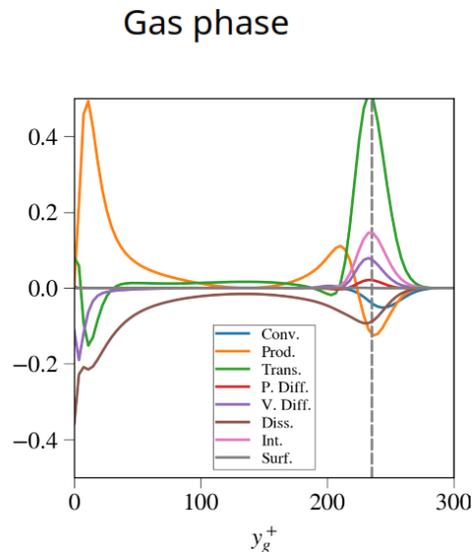
UDNS, Gas  $Re_\tau = 450$ , liquid  $Re_\tau = 250$ ,

- velocity:



DNS, Gas  $Re_\tau = 240$ , liquid  $Re_\tau = 120$ ,

- TKE budgets:

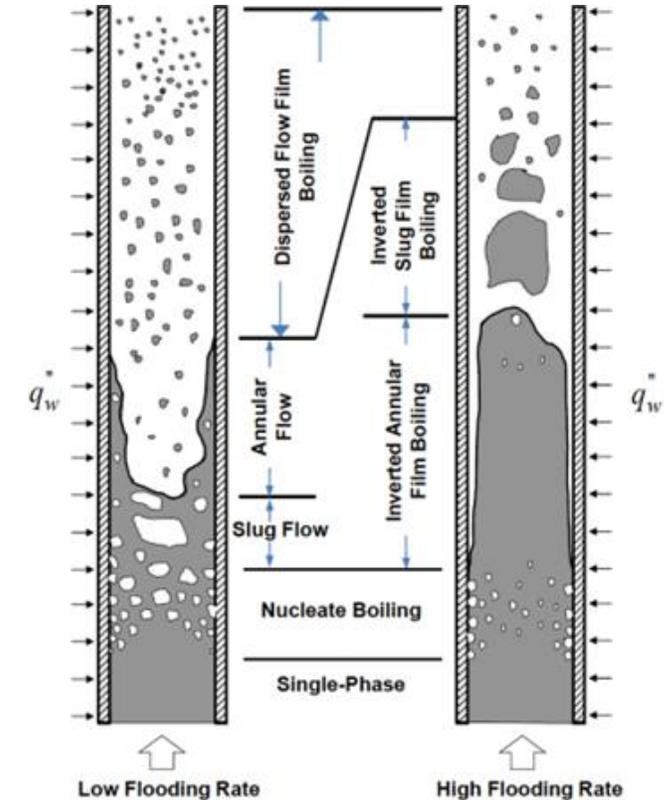


*A DNS example:*

- Stratified two-phase flow simulation
- 2<sup>nd</sup>-order accurate finite volume method
- PLIC VOF
- Purpose: knowledge and reference data for turbulence model development & validation
  - Near interface turbulence damping in RANS models

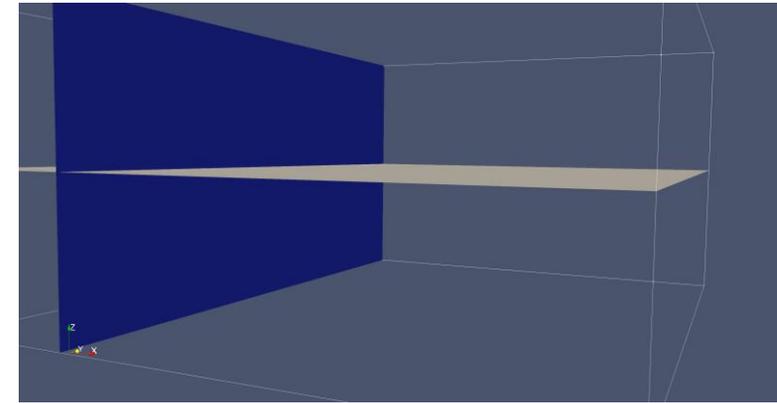
# HiFi two-phase simulations - Ambition

- Ambition must be to do
  - More complicated geometries
    - Subchannels, fuel assembly, spacers, mixing vanes,
  - More physics (coalescence, break-up, interfacial heat & mass transfer)
  - More realistic conditions
- To achieve this, we need
  - Better and more efficient numerical methods:
    1. 'Common' approach: structured grid, 2<sup>nd</sup>-order
    2. 'Refined' approach: unstructured grid, higher-order
  - Better / more accurate sub-grid models



# HiFi two-phase simulations – ‘Common’ approach

- Interface is second order at best → preference for high resolution over high order
  - *Does this rationale hold also for high-order discretization?*
- Structured staggered meshes:
  - VOF: geometric reconstruction relatively simple
  - Fast Poisson solvers:
    - FFTs (Schumann<sup>1</sup>)
    - Constant coefficient pressure equation (Dodd & Ferrante<sup>2</sup>)
    - *Large density ratios remains a challenge*
  - IBMs for complex geometries → *increases overhead and inaccuracies*
- Open-source examples: Basilisk, Briscola, CaNS (phase field, GPU)



B. J. Boersma.  
DLES13, Udine, Italy, 2022

131 mio cells

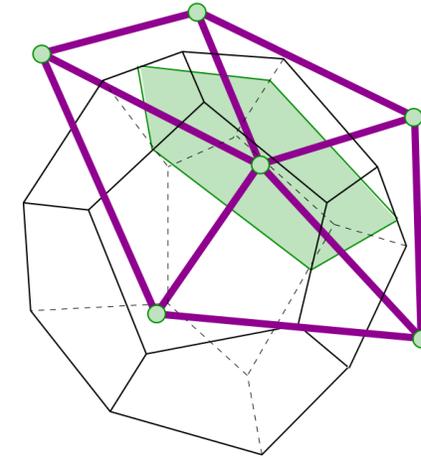
<sup>1</sup>Schumann, U., & Sweet, R. A. Journal of Computational Physics, 1998

<sup>2</sup>Dodd, M. S., & Ferrante, A. Journal of Computational Physics, 2014



# HiFi two-phase simulations – ‘Refined’ approach

- Complex geometries → Unstructured (collocated) grids
- Development of unstructured geometric VOF
  - Polyhedron truncation
    - Open-source toolboxes gVOF, VOFTools<sup>1</sup>
  - Unsplit advection<sup>2</sup>
    - More general, more complicated, not necessarily conservative
- Higher-order
  - e.g., implementation of LS in Nek5000/NekRS,



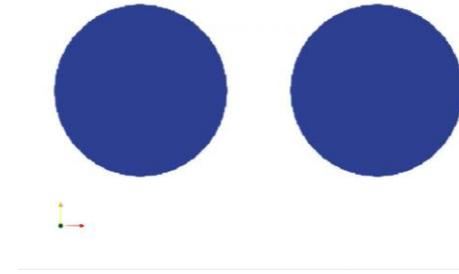
Truncation of an arbitrary polyhedron, Lopez & Hernández<sup>1</sup>

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López, J., & Hernández, J.. *Computer Physics Communications*, 2022  
 Marić, T., Kothe, D. B., & Bothe, D. *Journal of Computational Physics*, 2020.

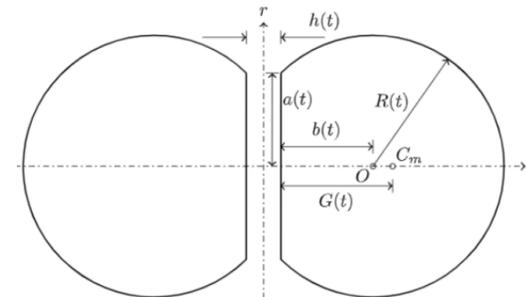
# HiFi two-phase simulations – sub-grid models

- Needed for:
  - Coalescence: avoid numerical coalescence, film drainage model
  - Break-up
  - Interfacial heat transfer
  - Nucleate boiling: nucleation, micro-layers
- Small-scale trouble: surface tension and viscous effects dominate → (Semi-)analytical solutions?<sup>1</sup>
- This motivates more modeling, not more resolution



e.g. bubble coalescence:

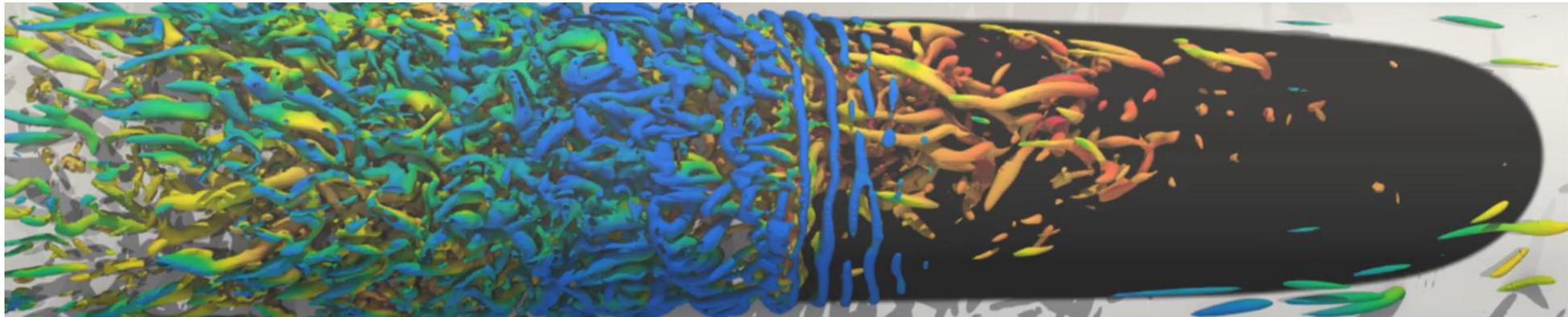
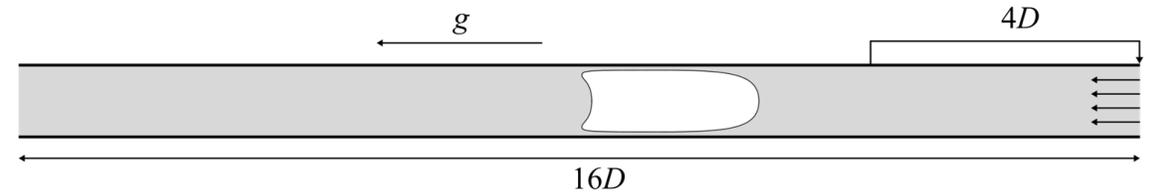
- scales thin film drainage process  $O(\text{few tens nm})$
  - much smaller scale than DNS mesh resolution
- Semi-analytical sub-models required



<sup>1</sup>G. Tryggvason et al. *Physics of Fluids*, 2013.

# HiFi two-phase simulations – potential

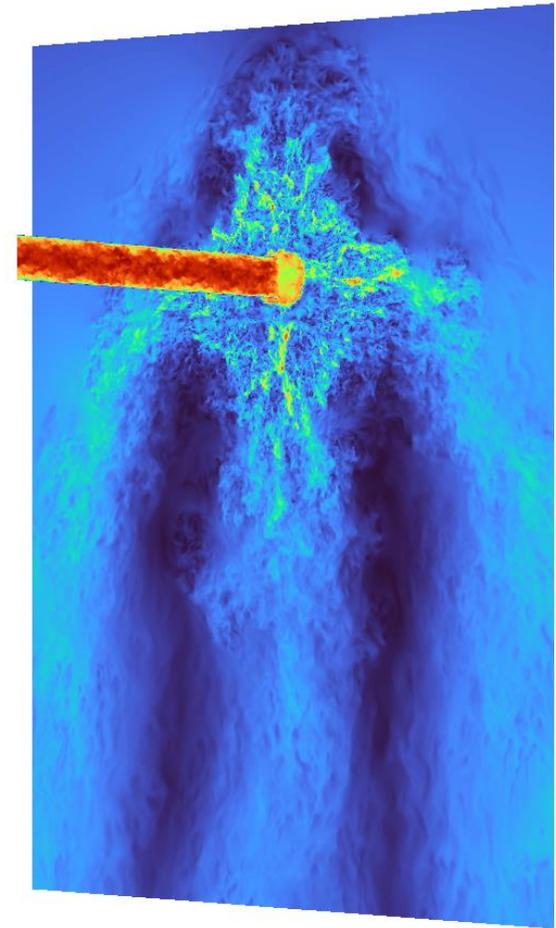
- HiFi two-phase simulations have a large potential
  - more fundamental insight
    - Separate effects
    - At conditions that are experimentally difficult
- Example of co-current Taylor bubble flow simulations with Basilisk



Turbulent Taylor bubble flow with  $\lambda_2$  iso-surfaces colored by streamwise velocity

# Concluding remarks

- Single-phase simulations in complex geometries:
  - DNS: possible, e.g. with SEM
  - LES: anisotropic models most needed for wall resolved simulations?
  - LES: implicit filtering → how to predict the correct friction velocity?
- Two-phase HiFi simulations:
- Ambition: 1) complex geometries, 2) more physics, 3) realistic conditions
- To achieve this, we must
  - Improve numerical methods
    - 'Common' approach: structured grid, 2<sup>nd</sup>-order
    - 'Refined' approach: unstructured grid, higher-order
  - Develop better sub-grid models



Mathur et al. *IJHMT*, 2024



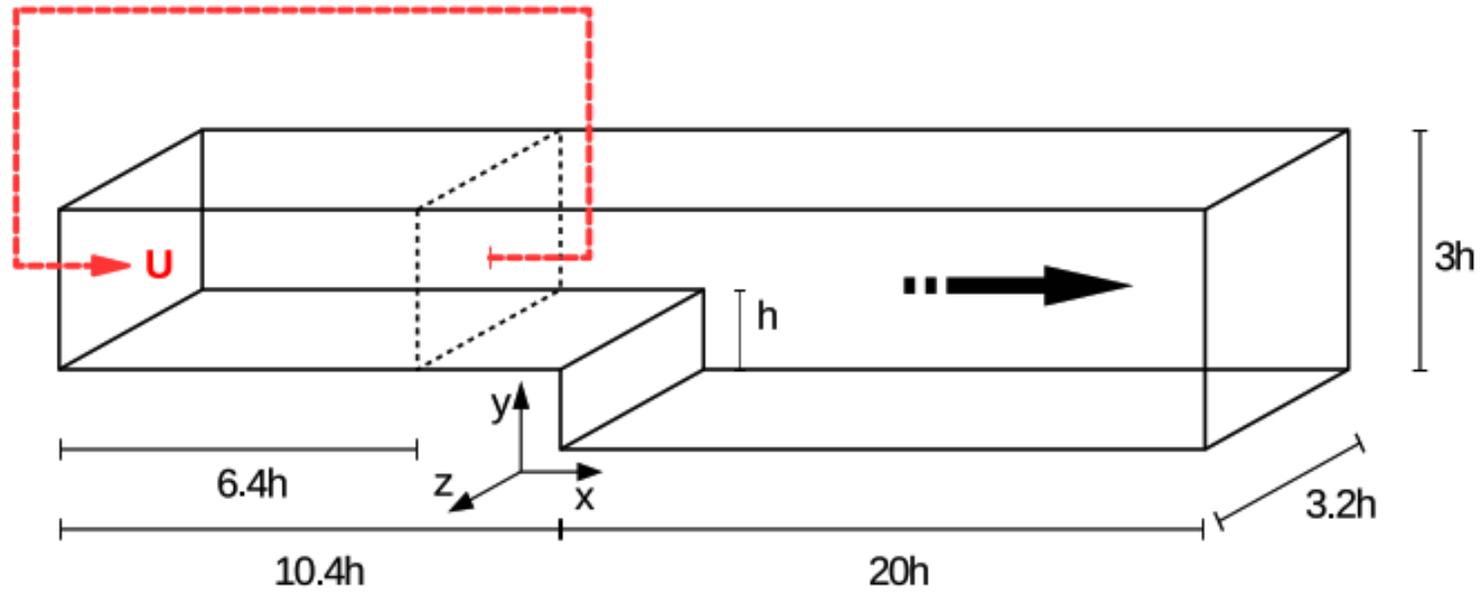
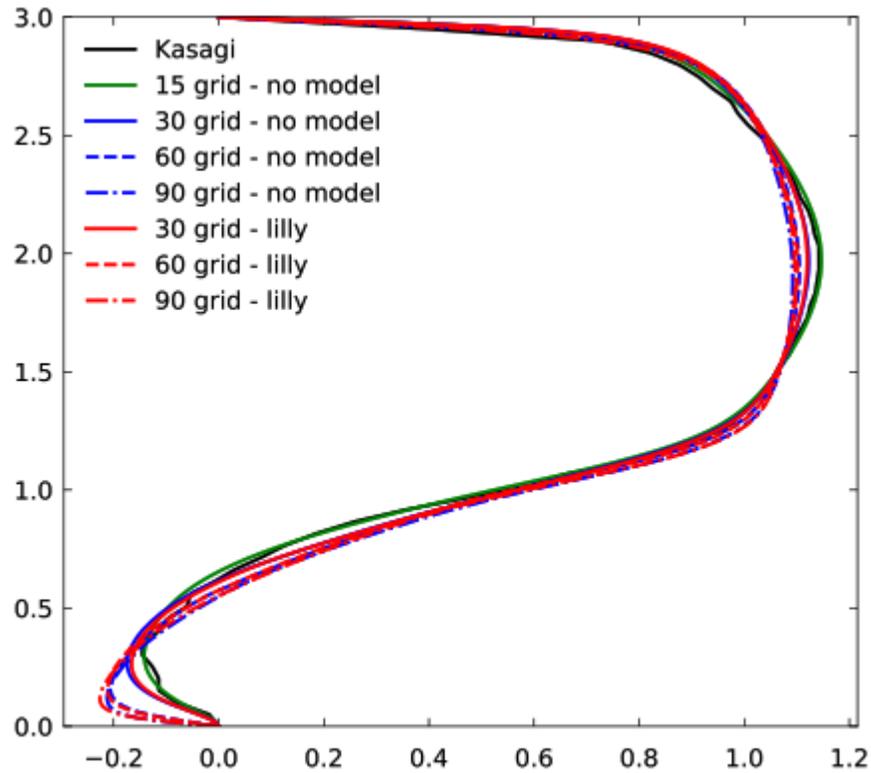


Figure 1: Backward facing step computational domain

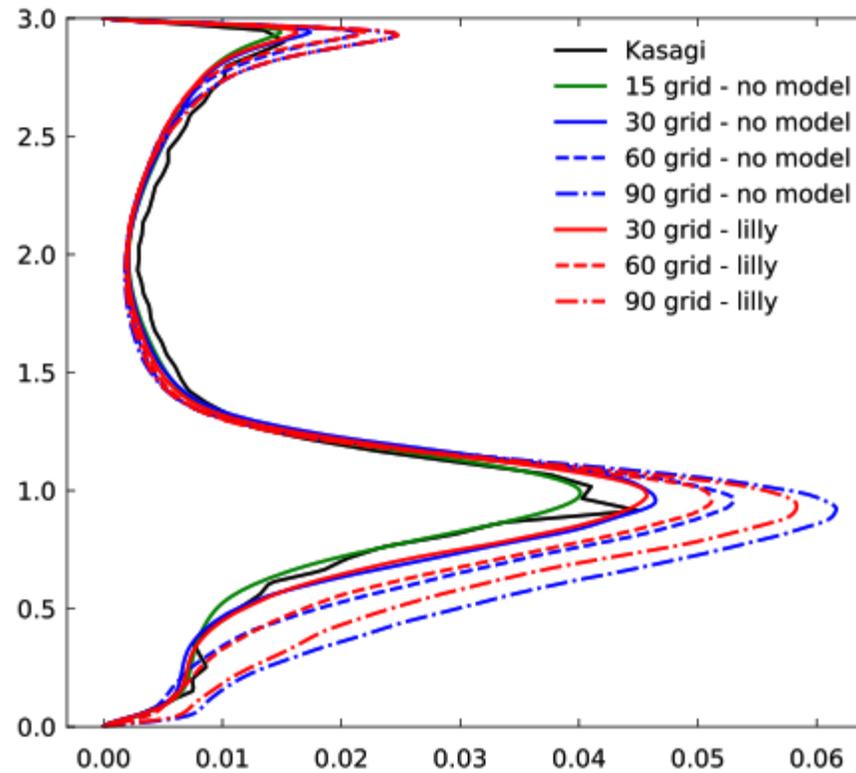
Table 3: Summary of the computational grids used for the present analyses.  $y_{wall}^+$  is the non-dimensional distance of the first cell center normal to the wall, whereas  $\Delta y_{bulk}^+$  represents the non-dimensional wall normal cell size in the center of the channel. A stretching ratio of 1.07 is used in the wall normal direction in order to stretch the cells from  $y_{wall}^+$  to  $\Delta y_{bulk}^+$ . In the streamwise direction, a stretching ratio of 1.05 is used in order to refine the cells from  $\Delta x_{max}^+$  to  $\Delta x_{min}^+$  towards step, and subsequently coarsen the cells from  $\Delta x_{min}^+$  to  $\Delta x_{max}^+$  downstream of the step.  $\Delta z^+$  represent the non-dimensional cell sizes in the spanwise direction.  $N_{total}$ , represents the total number of cells. The size of the computational domain are specified in Fig. 1.

Name	$\Delta x_{min}^+$	$\Delta x_{max}^+$	$\Delta y_{min}^+$	$\Delta y_{max}^+$	$\Delta z^+$	$r_{exp}$	$Re_{\tau}$	$N_{total}$
90-grid	5	90	0.5	30	30	1.05	180	1.144675e06
60-grid	5	60	0.5	20	20	1.05	180	2.122332e06
30-grid	5	30	0.5	10	10	1.05	180	6.849729e06
15-grid	5	15	0.5	7.5	7.5	1.05	180	1.728374e07

# BFS, $x/h = 2$

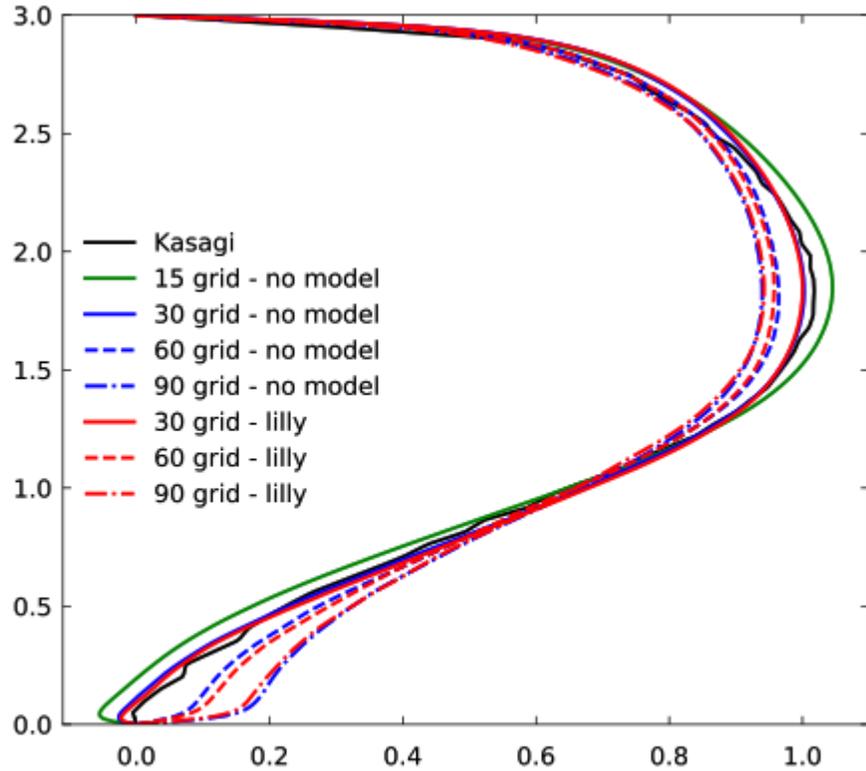


(a) Mean streamwise velocity  $\bar{U}/U_{bulk}$

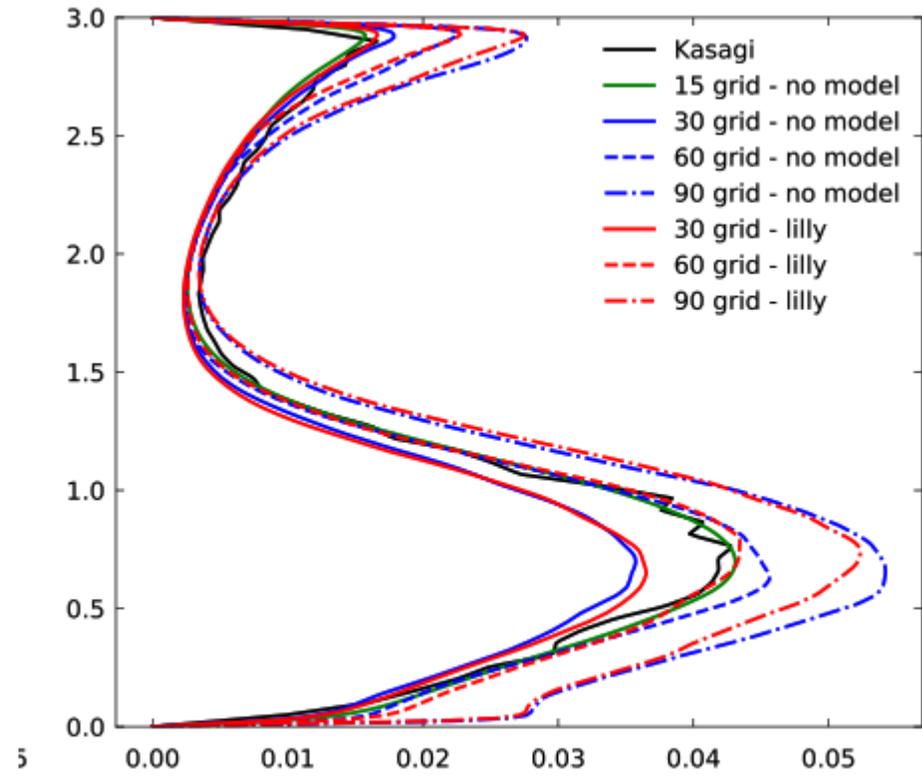


(f) Turbulent kinetic energy  $\bar{k}/U_{bulk}^2$

# BFS, $x/h = 6$



(a) Mean streamwise velocity  $\bar{U}/U_{bulk}$



(f) Turbulent kinetic energy  $\bar{k}/U_{bulk}^2$

BFS